



# Introduction to Computational Fluid Dynamics (CFD)

## Lecture 1, 2 (Part 2)

# Agenda – Lecture 1

- Approaches to solving engineering problems
- CFD (What? Why? How?)
- Fundamental equations

# Investigation approaches

## Experimental investigation

- Most reliable information
- Can use both full-scale and small-scale tests
- Not free from errors

## Theoretical calculation

- Usually set of differential equations
- Solution exists only for a narrow range of practical problems

## Numerical calculation (CFD)

- Finite number of domain elements (discretization)
- Set of algebraic equations
- Solution exists almost for each practical applications

# Strengths and weaknesses of theoretical approach

## Advantages (Strengths)

- Speed
- Low cost
- Information completeness
- Ability to simulate both ideal and real conditions

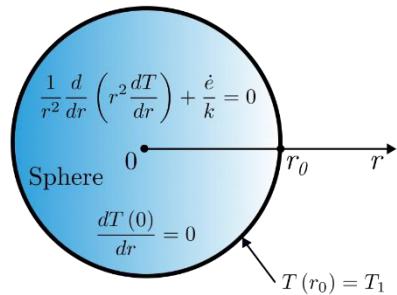
## Disadvantages (Weaknesses)

- Mathematical model
- Problem complexities

**Remember: Experiment leads,  
computation follows.**

# Why use numerical methods?

## Problem: Heat conduction in a sphere

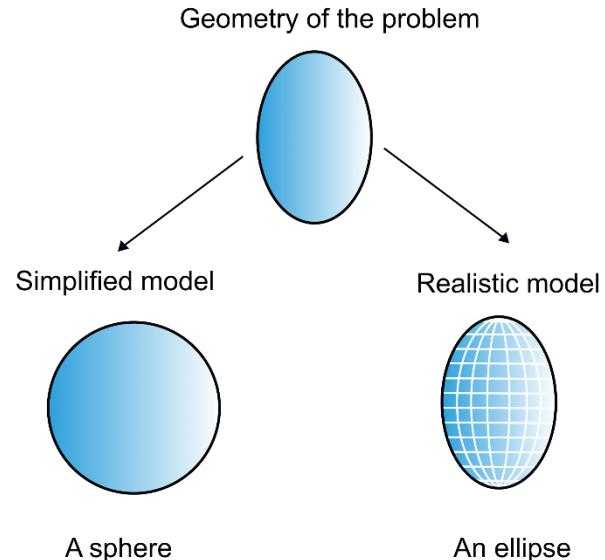


## Governing differential equation:

$$\frac{1}{r^2} \frac{d}{dr} \left( r^2 \frac{dT}{dr} \right) + \frac{\dot{e}}{k} = 0$$

## Solution:

$$T(r) = T_1 + \frac{\dot{e}}{6k} (r_0^2 - r^2)$$



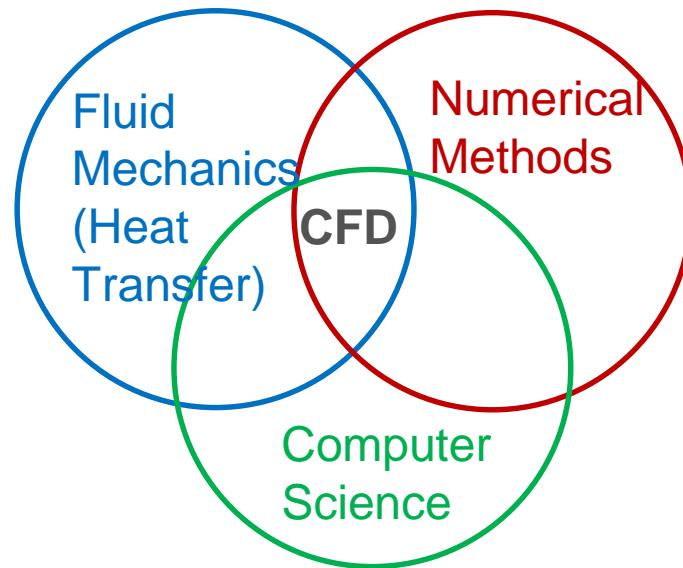
Exact (analytical) solution of model, but crude solution of actual problem

Approximate (numerical) solution of model, but accurate solution of actual problem

# Why prefer numerical approach to analytical?

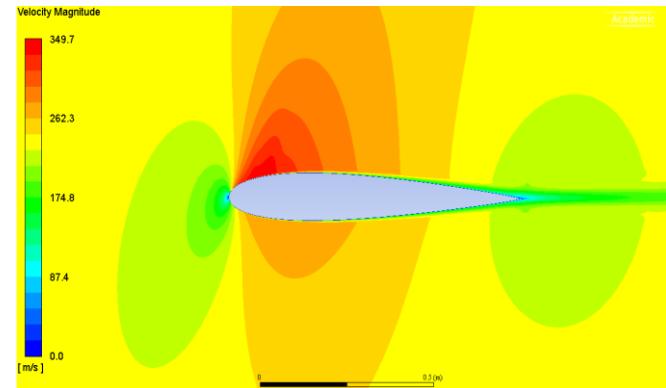
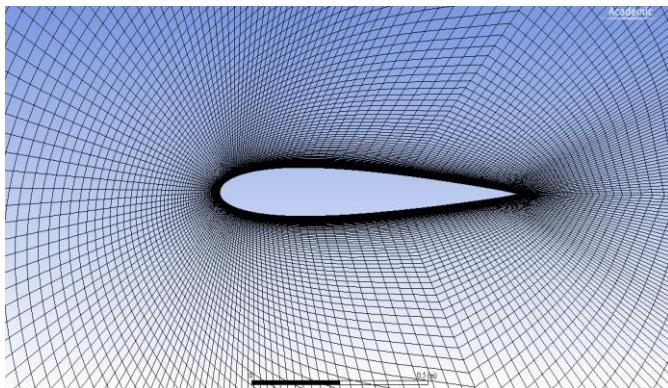
- **Limitations** (geometry, variable HTC, temperature dependent  $k$ , ...)
- **Better modelling** (“approximate” solution of a realistic model is usually more accurate than the “exact” solution of a crude mathematical model)
- **Flexibility** (parametric studies to answer some “what-if ” questions)
- **Complications** (even when analytical solutions are available, they can be intimidating)
- **Human nature** (ready availability of high-powered computers with sophisticated software packages)

# What is CFD?



# What is CFD?

- Numerical analysis and computers come into play.
- Differential equations → Algebraic equations → solution
- The application of CFD to practical problems is often limited by the computational power available.



# The black box idea

- Valid especially for commercial packages (Ansys Fluent, CFX, Star CCM+).
- More sophisticated codes are also available (OpenFOAM).
- User inputs (geometry, mesh, boundary conditions, material properties, solver settings).
- Turn the crank, and get the results (color pictures).



# Set of Fundamental Equations

## 1. Conservation of mass

$$\frac{\partial \rho}{\partial t} + \nabla \cdot [\rho \mathbf{U}] = 0 \quad (1)$$

## 2. Conservation of momentum

$$\frac{\partial [\rho \mathbf{U}]}{\partial t} + \nabla \cdot \{\rho \mathbf{U} \mathbf{U}\} = [\mathbf{f}_s] + [\mathbf{f}_v] \quad (2)$$

## 3. Conservation of energy

$$\frac{\partial (\rho e_{\text{total}})}{\partial t} + \nabla \cdot [\rho e_{\text{total}} \mathbf{U}] = \mathbf{f}_s \cdot \mathbf{U} + \mathbf{f}_v \cdot \mathbf{U} - \nabla \cdot [\mathbf{q}] \quad (3)$$

Note:

$$e_{\text{total}} = e_{\text{internal}} + \frac{1}{2} \mathbf{U} \cdot \mathbf{U} \quad (4)$$

$$\mathbf{q} = -k \nabla T \quad (5)$$

# Set of Fundamental Equations

## 1. Conservation of mass

$$\frac{\partial \rho}{\partial t} + \nabla \cdot [\rho \mathbf{U}] = 0 \quad (6)$$

## 2. Conservation of momentum

$$\frac{\partial [\rho \mathbf{U}]}{\partial t} + \nabla \cdot \{\rho \mathbf{U} \mathbf{U}\} = [\mathbf{f}_s] + [\mathbf{f}_v] = \nabla \cdot \boldsymbol{\sigma} + [\rho \mathbf{g}] = -\nabla p + \nabla \cdot \boldsymbol{\tau} + [\rho \mathbf{g}] \quad (7)$$

## 3. Conservation of energy

$$\frac{\partial (\rho e_{\text{total}})}{\partial t} + \nabla \cdot [\rho e_{\text{total}} \mathbf{U}] = \mathbf{f}_s \cdot \mathbf{U} + \mathbf{f}_v \cdot \mathbf{U} - \nabla \cdot [\mathbf{q}] = -\nabla \cdot [p \mathbf{U}] + \nabla \cdot [\boldsymbol{\tau} \cdot \mathbf{U}] + [\rho \mathbf{g}] \cdot \mathbf{U} + \nabla \cdot [k \nabla T] \quad (8)$$

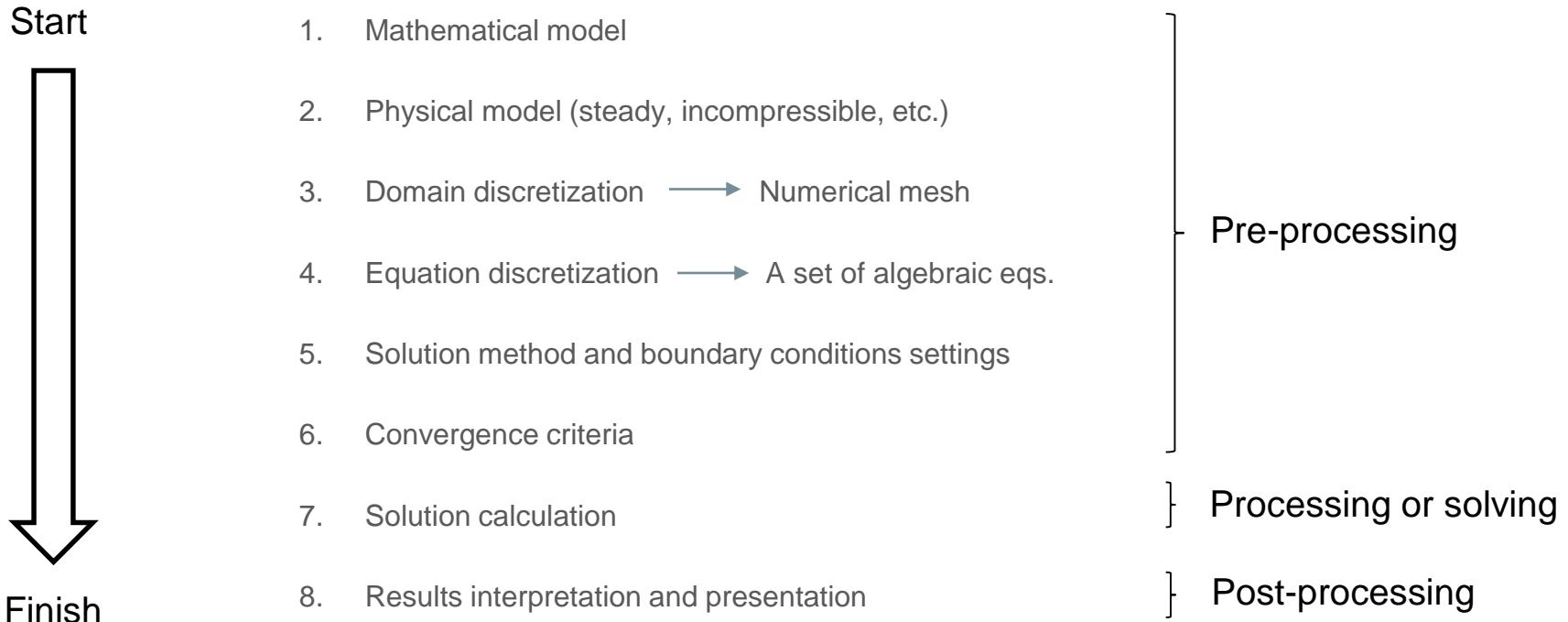
# Summary – Lecture 1

- CFD can be a handy tool (What? Why? How?)
- Governing equations: **mass**, **momentum** (Newton's 2nd law), and **energy** (1st law of thermodynamics)

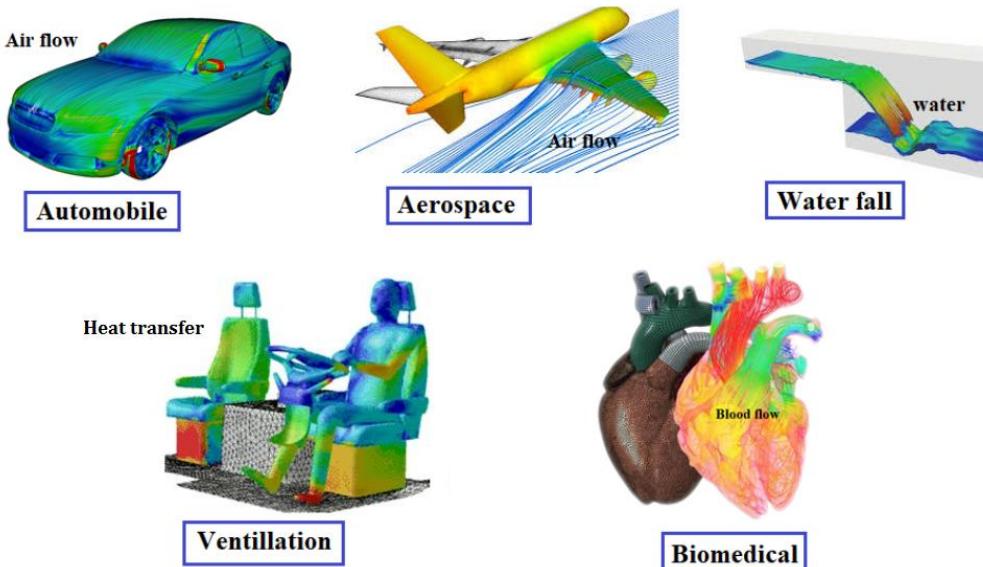
# Agenda – Lecture 2

- Numerical procedure
- Some CFD applications
- Turbulence in CFD

# Numerical Simulation Procedure



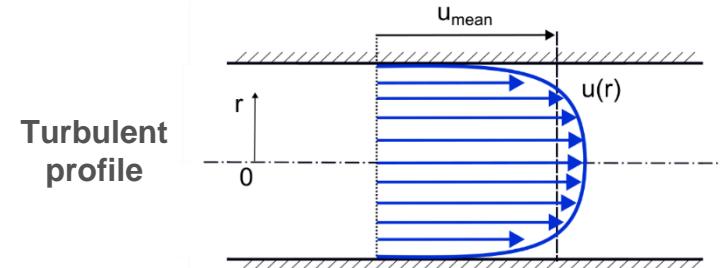
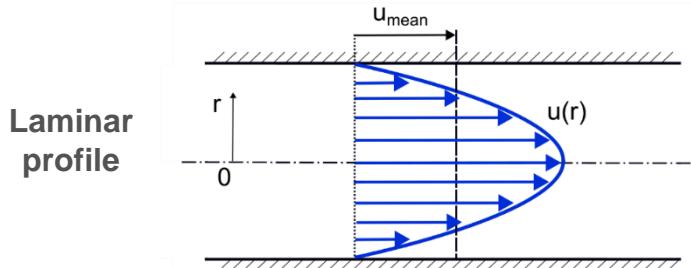
# Applications of CFD



Source: <https://cfdflowengineering.com/scope-of-cfd-modeling-career-and-job-opportunities/>

# Treatment of turbulence in CFD

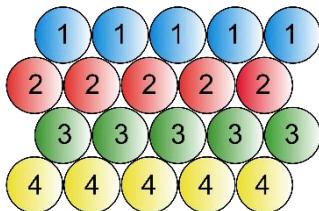
- Flow can be laminar, turbulent (more frequent), or transitional (complex to solve).
- Most flows in practice are turbulent.
- Laminar solutions are almost exact (Mesh resolution, BCs).
- Resolution of complex turbulent flows is challenging and not feasible these days.



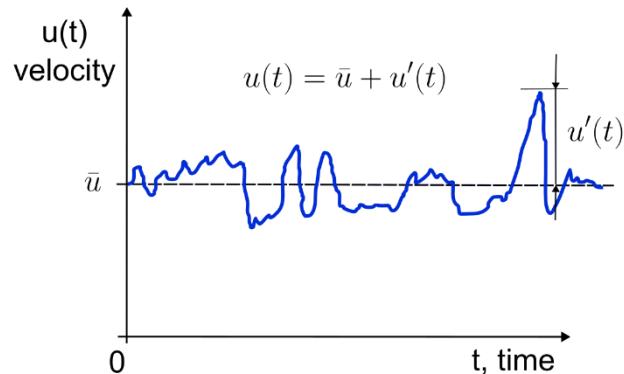
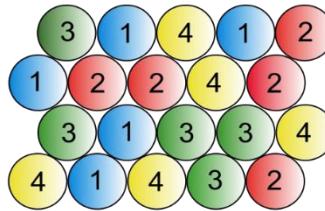
# Treatment of turbulence in CFD (2)

- Direct Numerical Simulation (DNS) is not useful for practical engineering problems.
- It would require a very fine mesh to capture all turbulent motions.
- Therefore, we must rely on experiments and empirical correlations.

Before turbulence



After turbulence



# Treatment of turbulence in CFD (3)

- Turbulent flows are characterized by random and rapid fluctuations of swirling regions (= eddies).
- We need to capture these turbulent structures somehow.
- One of the options how to do it is to get inspiration in molecular motion fluctuations.
- In laminar flows, the molecular viscosity causes the shear stress.
- In turbulent flows, this shear stress is still present and additional stresses arise from the turbulent fluctuations.

$$\tau_{\text{total}} = \tau_{\text{laminar}} + \tau_{\text{turbulent}}$$

(9)

# Treatment of turbulence in CFD (4)

- The laminar component of the total shear stress can be expressed as:

$$\tau_{\text{laminar}} = -\mu \frac{\partial \bar{U}}{\partial r} = -\mu \frac{\partial \bar{U}}{\partial y} \quad (10)$$

- In an analogous manner, we can express the turbulent component:

$$\tau_{\text{turbulent}} = -\mu_t \frac{\partial \bar{U}}{\partial r} = -\mu_t \frac{\partial \bar{U}}{\partial y} \quad (11)$$

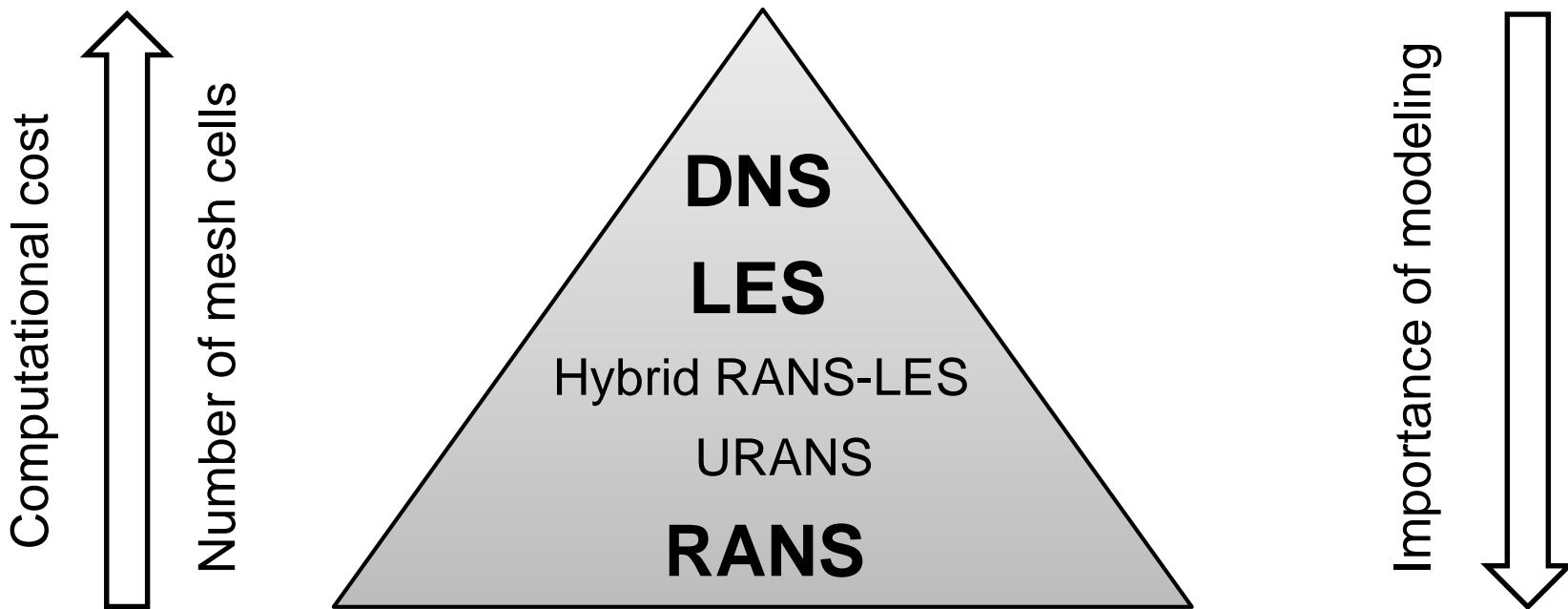
- The problem here is that we do **not have**  $\mu_t$ , which is not a material constant as  $\mu$ , but rather a property of turbulent flow.
- Note that the turbulent shear stress is often also expressed as:

$$\tau_{\text{turbulent}} = -\rho \bar{u}' \bar{v}' \quad (12)$$

# Treatment of turbulence in CFD (5)

- Most simulations require a model (a coarser mesh can be used).
- No universal model exists for all turbulent flows.
- Turbulence models aim to represent the effect of turbulence via some additional terms or equations.
- Models try to capture the mixing and diffusion caused by turbulent eddies.
- CFD results are only as good as the turbulence model used.

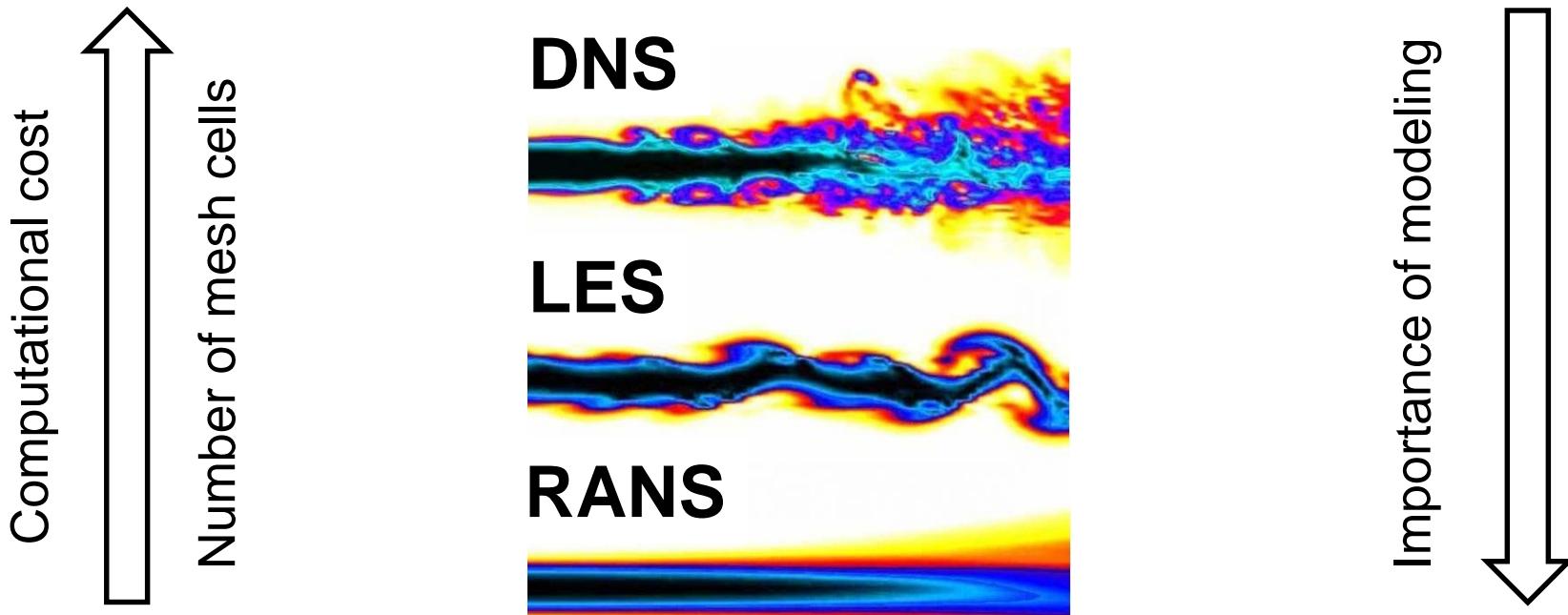
# Turbulence models in CFD



DNS - Direct Numerical Simulation  
LES - Large Eddy Simulation

RANS - Reynolds-Averaged Navier-Stokes  
URANS - Unsteady (transient) RANS

# Turbulence models in CFD (2)

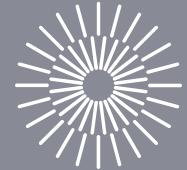


DNS - Direct Numerical Simulation  
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# Summary – Lecture 2

- Computational process step by step (from pre- to post-processing)
- Some applications
- Treating turbulence phenomena



Thank you.